

Robustly estimating and compensating uncertain oscillatory output

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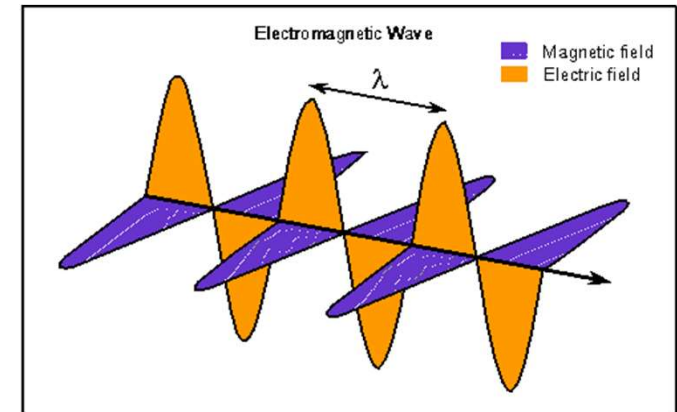
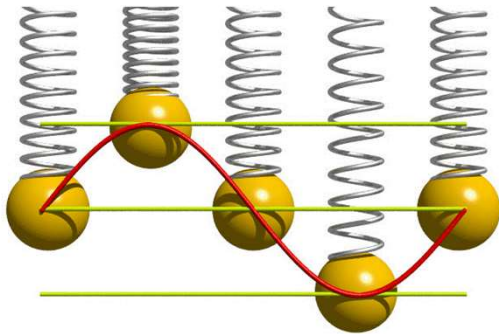
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Guest seminar

Loughborough University

- Robust frequency estimation



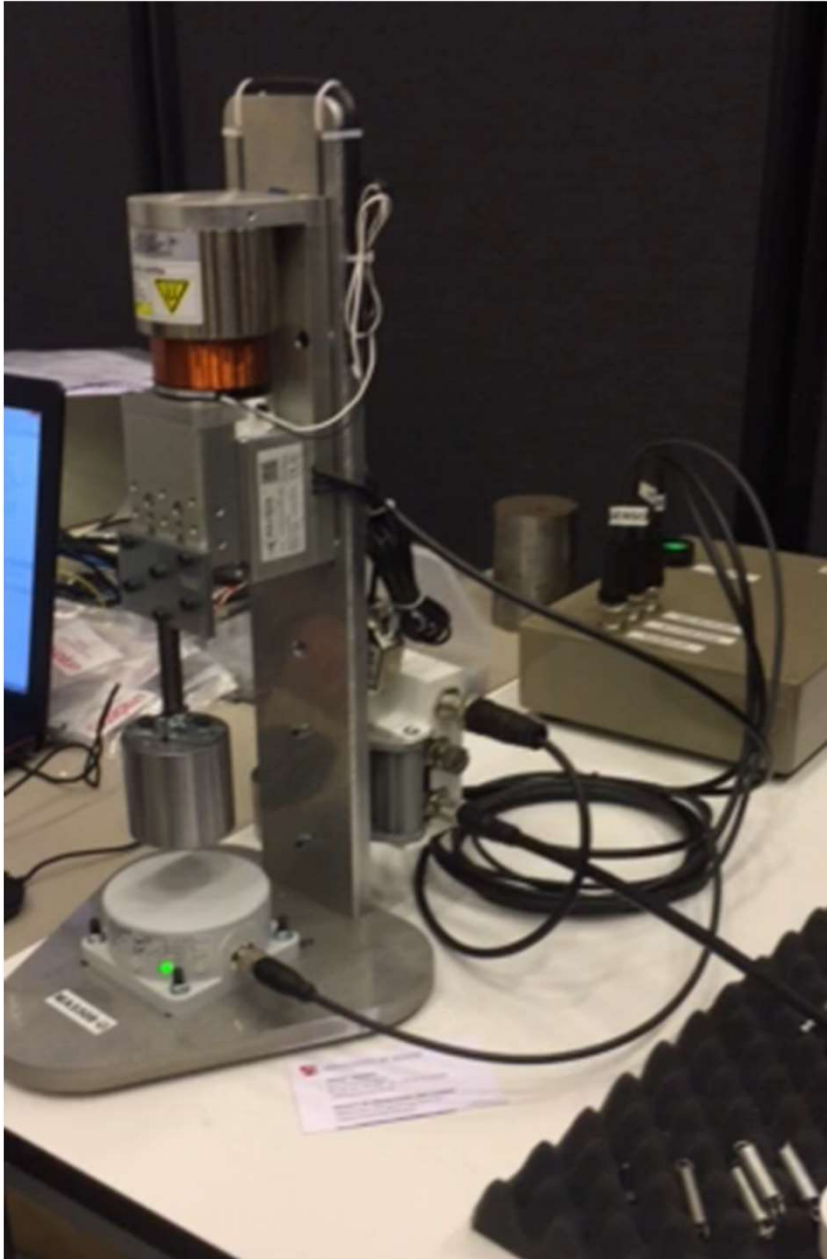
Measurement, analysis, monitoring of oscillatory quantities in almost all types of the systems:
 mechanical vibrations, electric & power signals, electromagnetic waves, bio-medical signals

General issues with oscillatory output signals: bias, noise, time-varying parameters

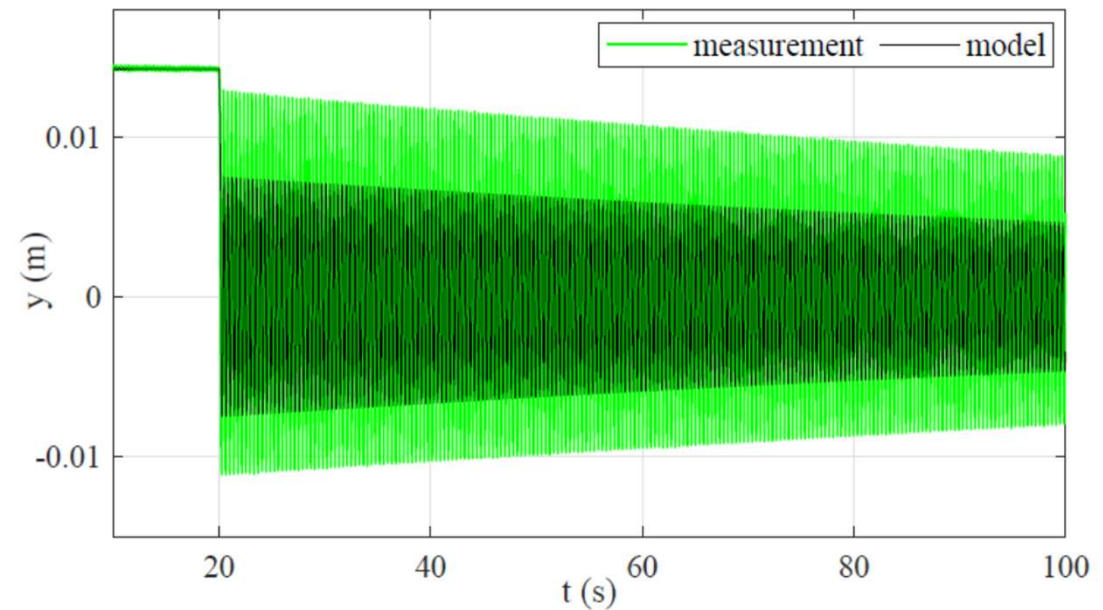
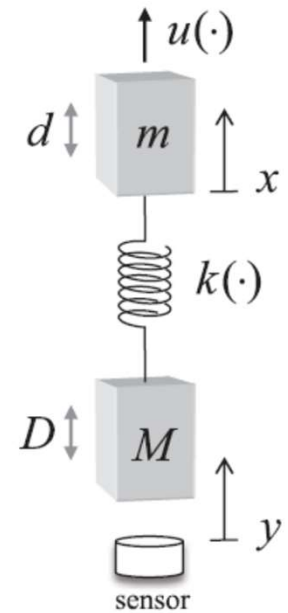
$$y(t) = Y_0 + Y(t) \sin(\omega(t) \cdot t + \varphi) + \eta(t)$$

ref. [5]

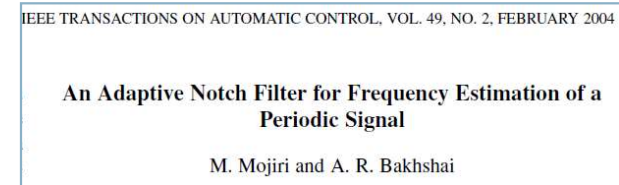
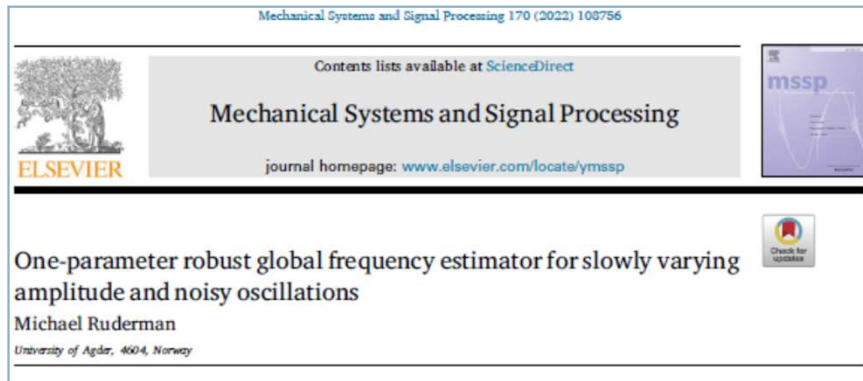
Numerous approaches from last three decades (including various PLLs), also the global estimator



Two-mass actuator-load system (anno 2020) for oscillations analysis & control prototyping



Robust frequency estimation



- One-parameter robust frequency estimator [1], motivated by [5], for noisy harmonic signals with a slowly-varying amplitude

$$\sigma(t) = k(t) \sin(\omega_0 t) + \eta(t) \quad \text{eq. (1)}$$

- ✓ Amplitude variations are slow (comparing to ω_0), and k appears as a ‘frozen’ term in analysis
- ✓ Measurement noise is zero-mean with a constant power spectral density (PSD), i.e. white-noise

$$PSD\{\eta(\omega)\} = \text{const} \equiv p$$

- Original global frequency estimator [4], [5] (adaptive notch filter)

$$\ddot{x}(t) + 2\zeta\theta\dot{x}(t) + \theta^2 x(t) = \theta^2 \sigma(t), \quad \dot{\theta} = -\gamma x(t) (\theta^2 \sigma(t) - 2\zeta\theta\dot{x}(t))$$

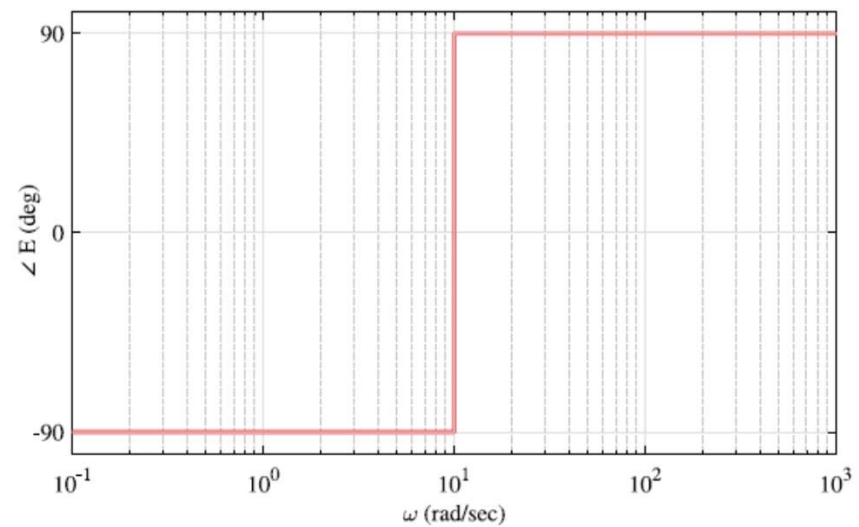
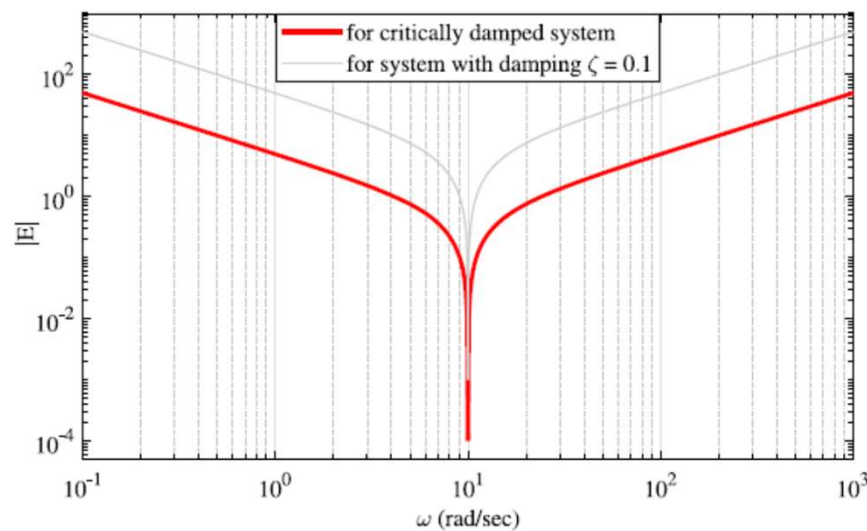
- One-parameter robust frequency estimator [1], eqs. (4)-(6)

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\theta^2 & -2\theta \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 2\theta \end{bmatrix} \sigma, \quad y = \begin{bmatrix} 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}, \quad \dot{\theta} = -\gamma \text{sign}(x_1)(\sigma - y)$$

at the angular frequency $\theta = \omega_0 \Rightarrow e = \sigma - y \rightarrow 0$

$$G(j\omega) = \frac{y(j\omega)}{\sigma(j\omega)} = C^T (j\omega I - A)^{-1} B \Rightarrow e(j\omega) = (1 - G)G^{-1}y(j\omega)$$

$$E(j\omega) = \frac{e(j\omega)}{y(j\omega)} = (1 - G(j\omega))G(j\omega)^{-1}$$



Theorem 1. The frequency estimator (4)–(6) is global for (1) and converges asymptotically as $\theta(t) \rightarrow \omega_0$ for $t \rightarrow \infty$, regardless of the $\theta(0) > 0$ initialization, provided the small adaptation gains $\gamma > 0$ and slowly varying amplitudes $k(t)$. The frequency-estimation error $\varepsilon(t) = \omega_0 - \theta(t)$ converges uniformly and exponentially in terms of

$$|\varepsilon(t_2)| < \alpha |\varepsilon(t_1)| \exp(-\beta(t_2 - t_1)) \quad (9)$$

for some $\alpha > 0$ and $\forall t_2 > t_1$. The exponential rate of convergence is independent of $\eta(t)$ noise and as follows:

$$\beta = 0.5 \gamma k \omega_0^{-1} + \delta, \quad (10)$$

where δ is a small positive constant independent of γ , k , ω_0 .

- Sketch of the proof (for more details see [1])

Let $0 < \theta(0) < \omega_0$ be an arbitrary initialization

$$y(t) = b \sin(\omega_0 t + c) \quad \text{where } b = k |G(j\omega_0)|$$

$$\Rightarrow x_1(t) = -\frac{b}{\omega_0} \cos(\omega_0 t + c) = -\frac{b}{\omega_0} \sin(\omega_0 t + c + \pi/2)$$

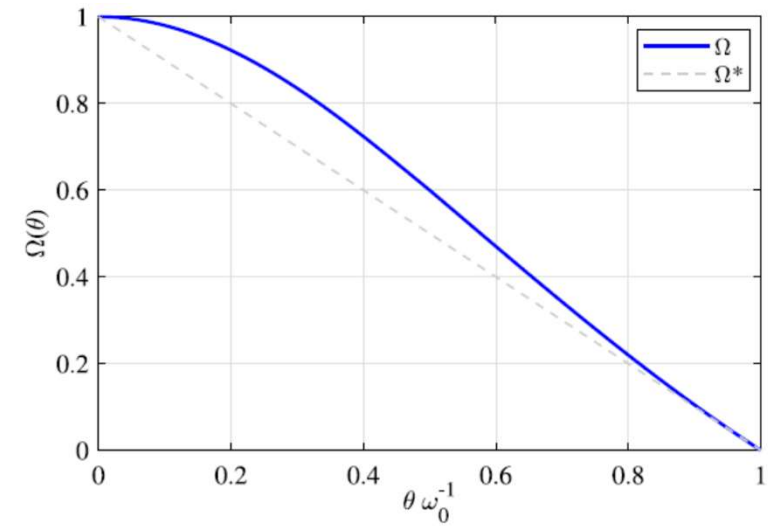
$$\Rightarrow e(t) = a \sin(\omega_0 t + c + \pi/2) \quad \text{where } a = b |E(j\omega_0)|$$

$$\dot{\theta} = \gamma k |G(j\omega_0)| \left| \frac{1 - G(j\omega_0)}{G(j\omega_0)} \right| |\sin(\omega_0 t + c + \pi/2)|$$

It can be seen that for all $\gamma, k > 0$ the $\text{sign}(\dot{\theta}) = +1$ as long as $\theta(t) < \omega_0$

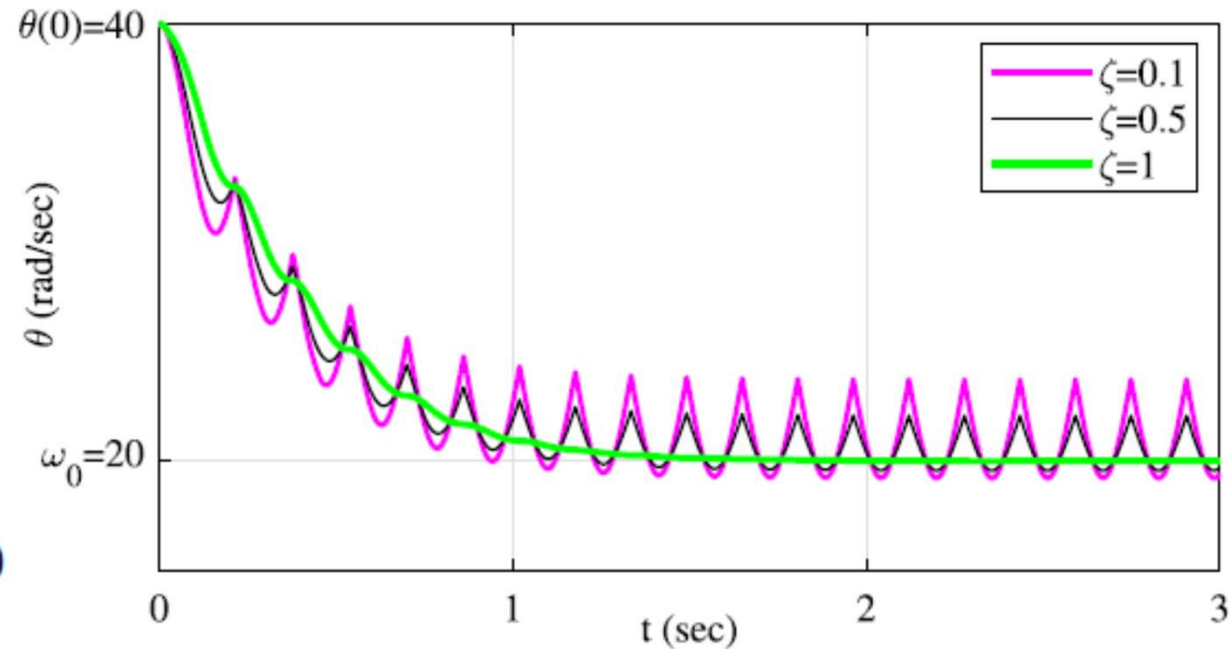
$$|G(j\omega_0)| \left| \frac{1 - G(j\omega_0)}{G(j\omega_0)} \right| = \frac{|\theta^2 - \omega_0^2|}{\theta^2 + \omega_0^2} \equiv \Omega(\theta)$$

$$\Omega^*(\theta) = -\theta \omega_0^{-1} + 1 \quad \Rightarrow \quad \dot{\theta}^* = 0.5 \gamma k (-\theta^* \omega_0^{-1} + 1) \quad \Rightarrow \quad \dot{\theta}(t) + 0.5 \gamma k \omega_0^{-1} \theta(t) = 0.5 \gamma k \omega_0^{-1} \cdot \omega_0$$



- Optimality without damping parameter (i.e. for $\zeta=1$)

estimator ($\gamma = 100$, $\omega_0 = 20$ rad/s)

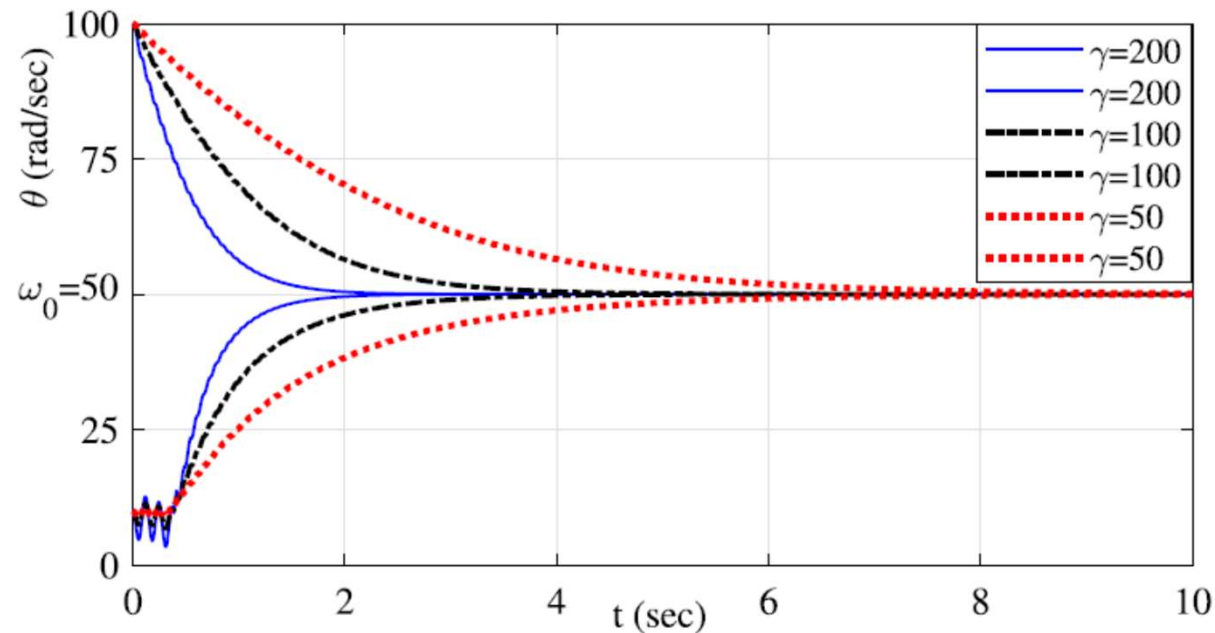


- Convergence speed for various adaption gains γ

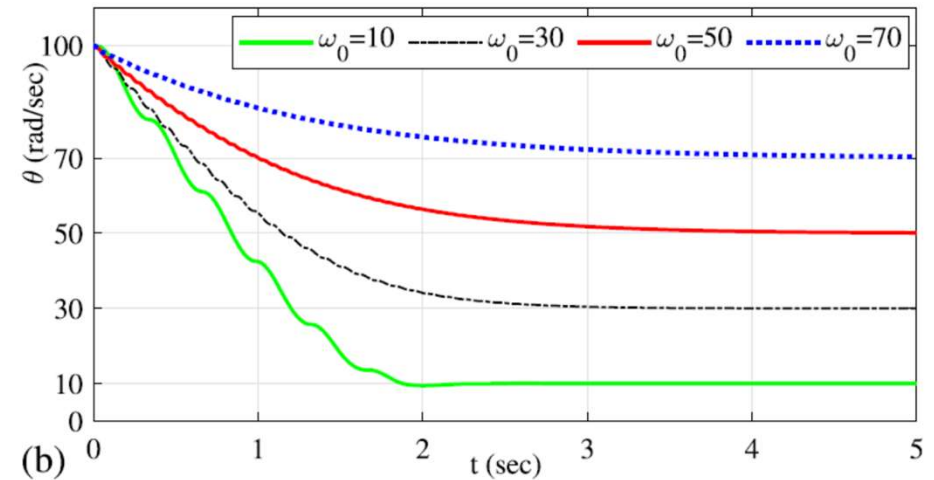
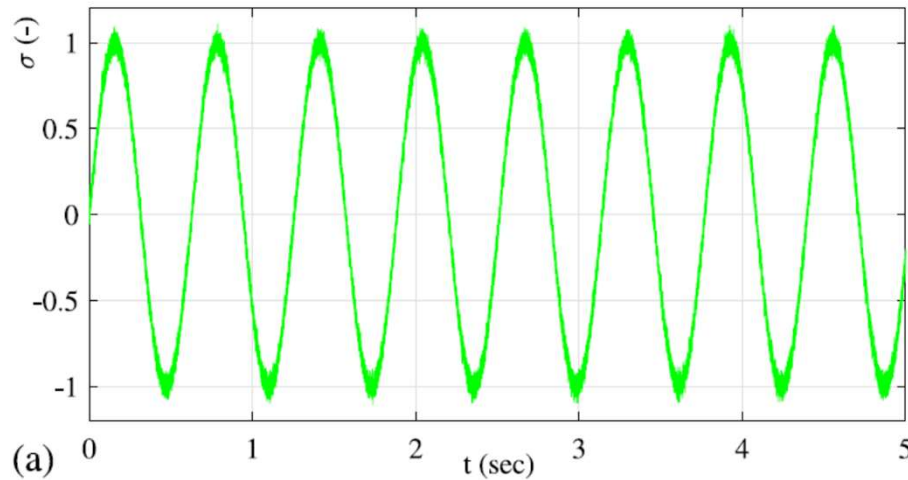
estimator

$\omega_0 = 50$ rad/s for $\sigma(t) = \sin(\omega_0 t)$

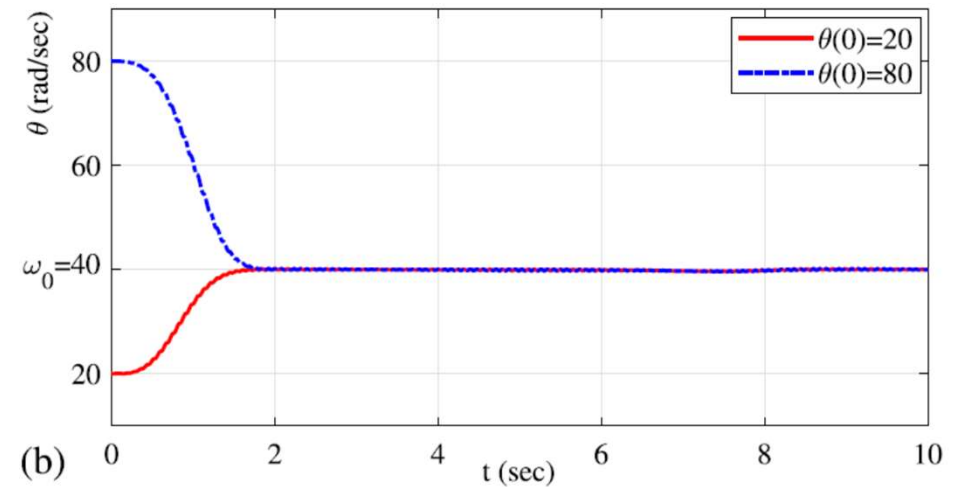
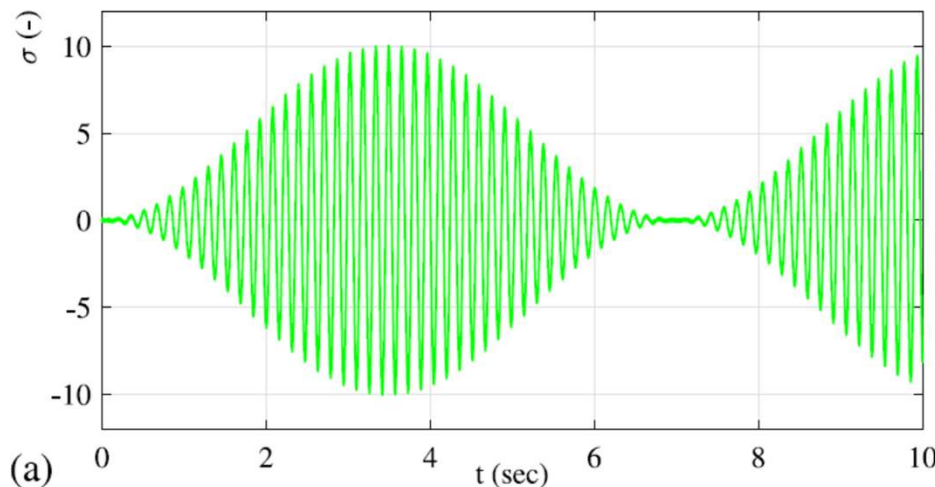
$\gamma = \{200, 100, 50\}$



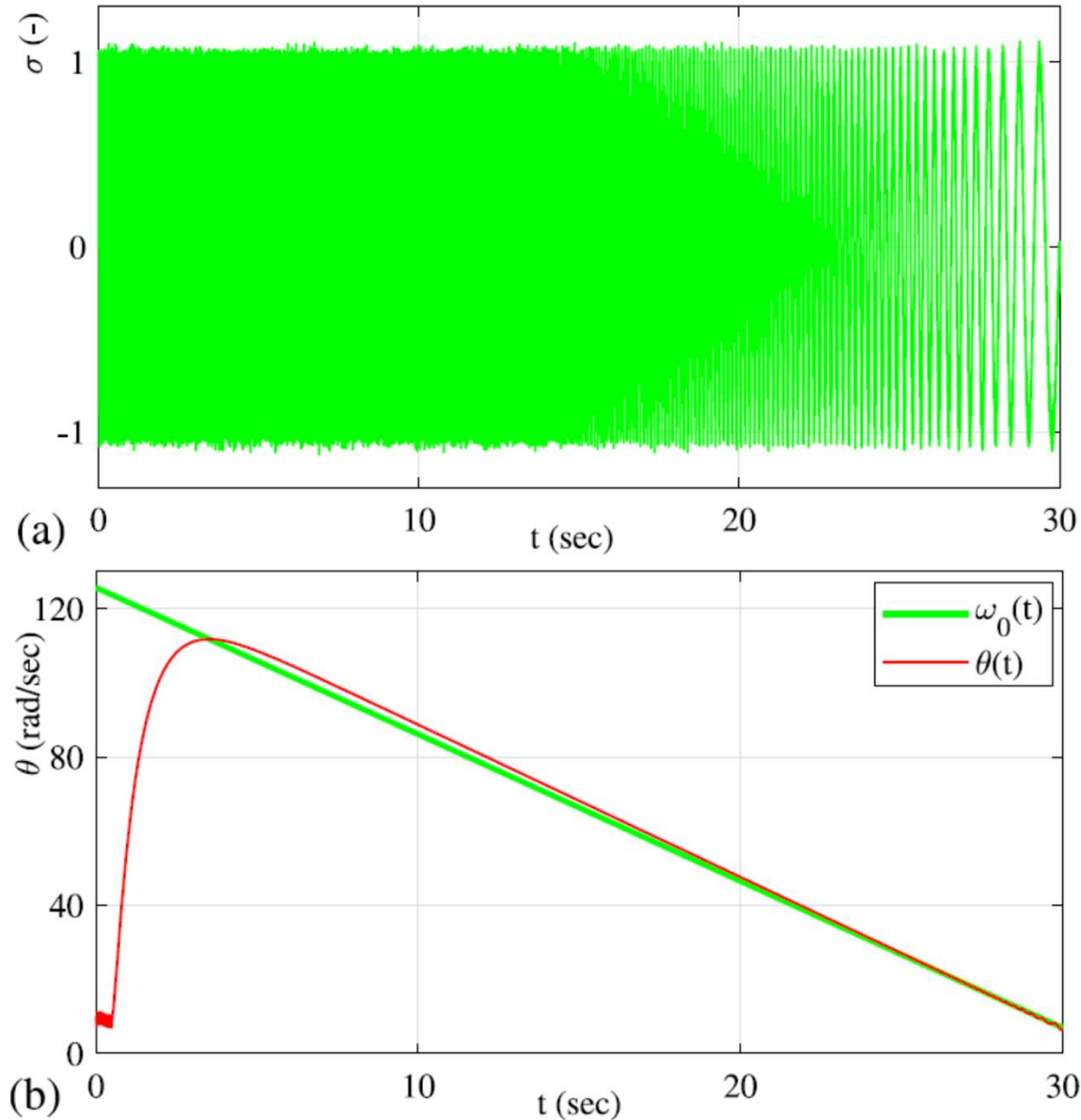
- Noisy signal, constant amplitude, various frequencies



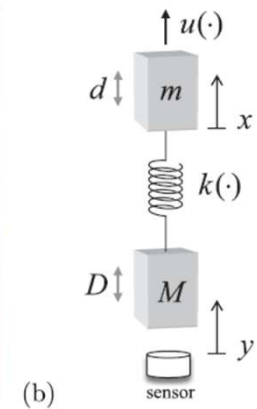
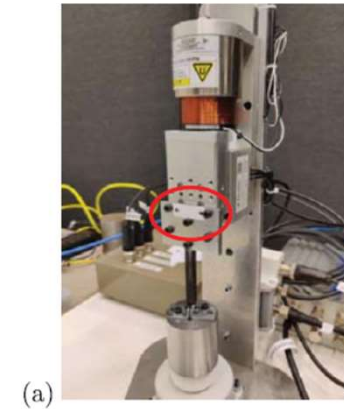
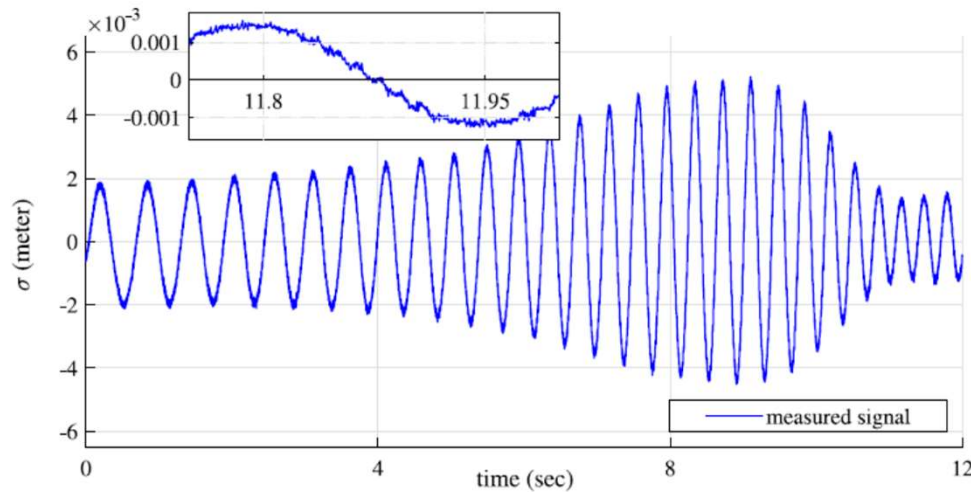
- Noisy signal, time-varying amplitude



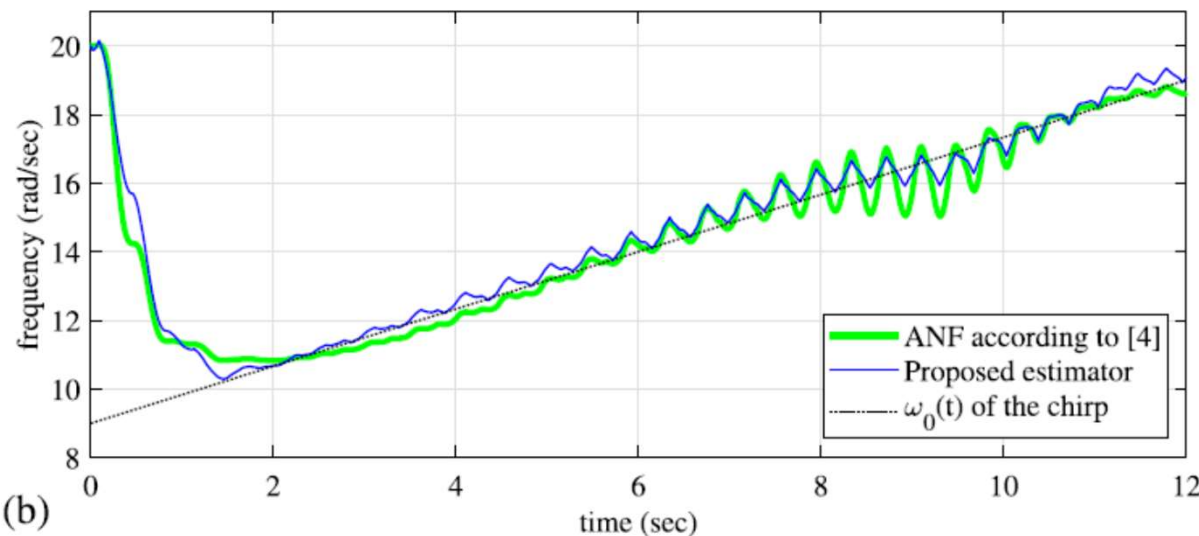
- Noisy signal, time-varying frequency (down-chirp signal)



- Noisy & biased signal with time-varying amplitude & frequency



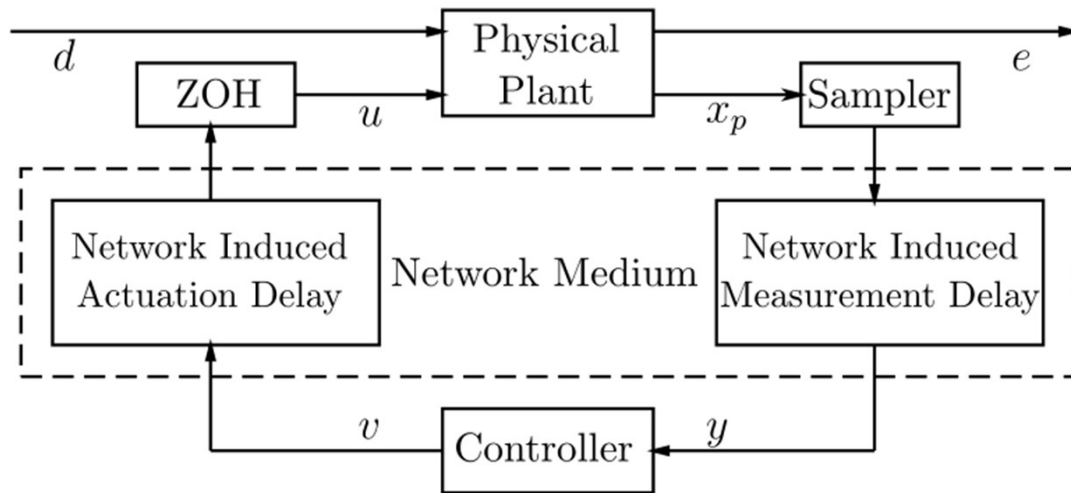
- Comparison with the original global estimator [4]


Table 1

Estimators' evaluation setting.

Set parameter	ANF according to [4]	Proposed estimator
$\theta(0)$ (rad/s)	20	20
γ	$4e4$	$2e4$
ζ	{0.7, 1, 1.3}	{0.7, 1, 1.3}

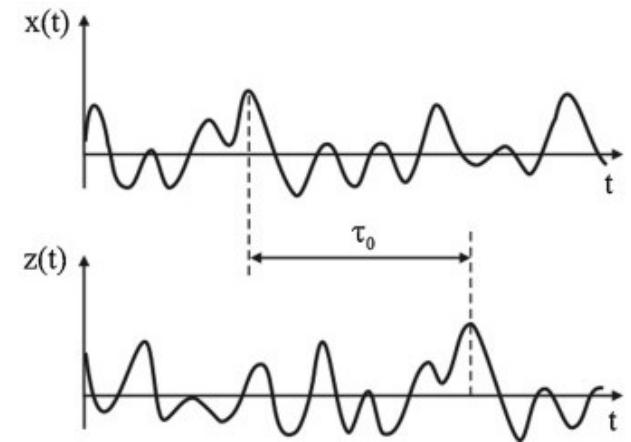
(Adaptive) time delay-based compensation



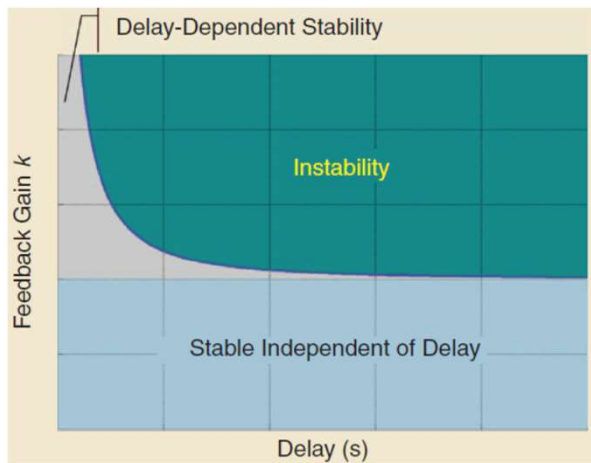
Shift of the time series by an (evtl. time-varying) factor

$$y(t) = x(t - \tau_0)$$

$$\tilde{G}(s) = G(s) \exp(-s\tau_0)$$



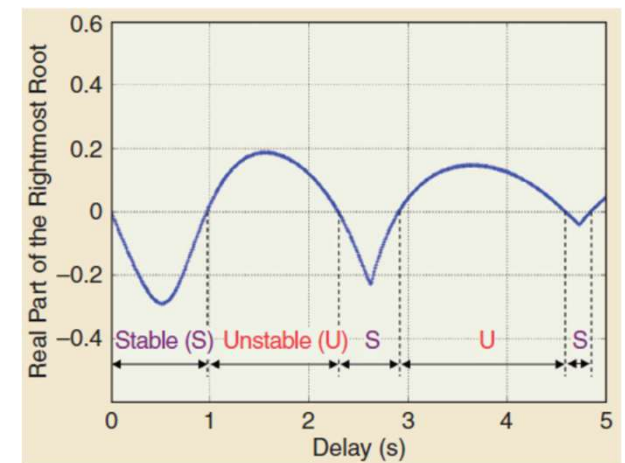
Usually, source of instability in feedback control systems
 ⇒ need for careful analysis and robust control design



$$\arg[\exp(-j\omega\tau_0)] = -\omega\tau_0$$

However, purposefully inserted time delays can be used for stabilization

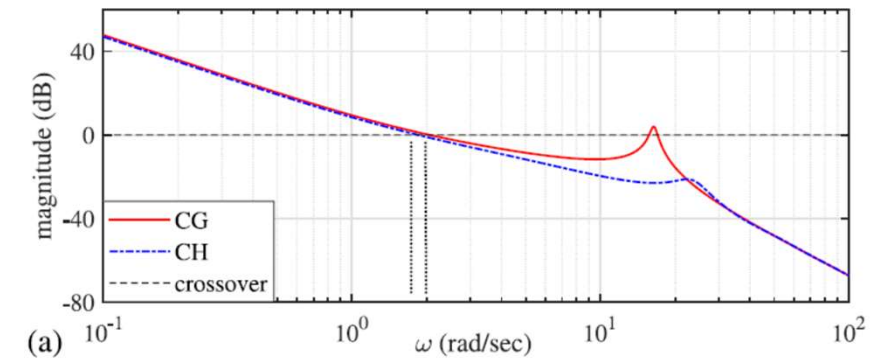
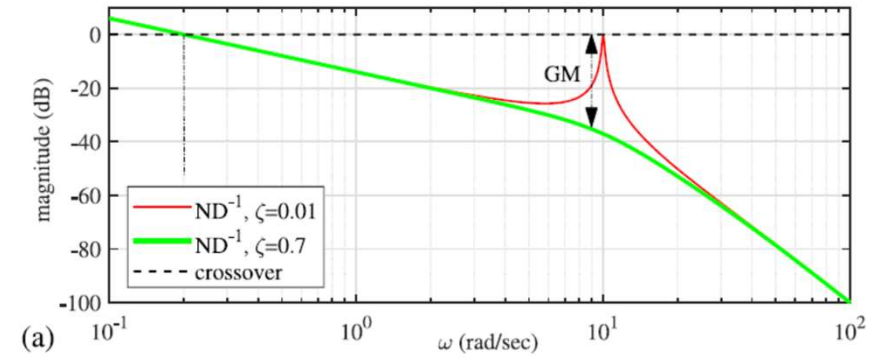
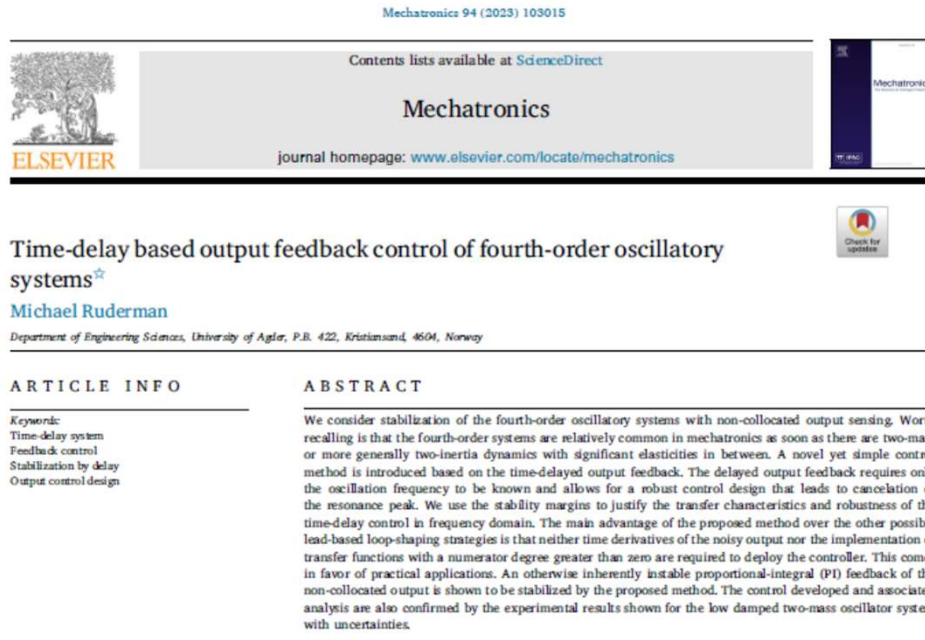
$$u = f(x(t), x(t - T))$$



ref. [6]

ref. [6]

- Original approach, initially anno 2021 [7], and elaborated in [2]



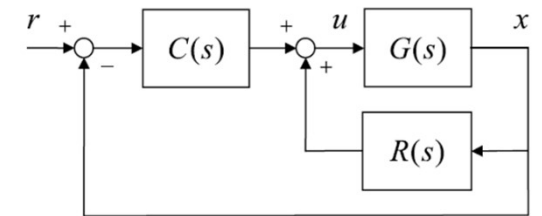
Research questions:

- Robust stabilization of non-collocated passive load
- Use of output feedback only \Rightarrow time-delay based control

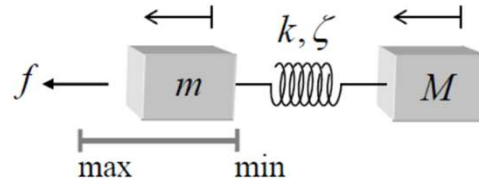
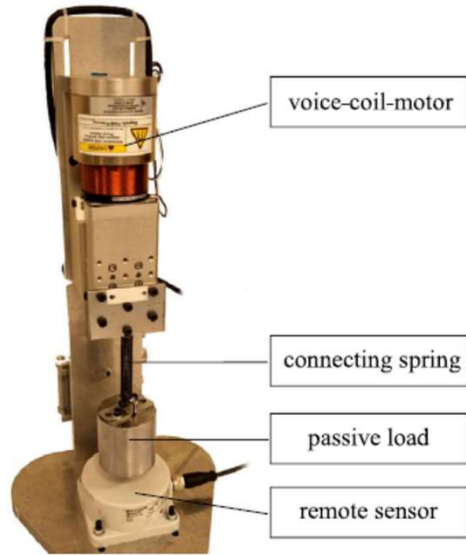
$$C(s)G(s) = K_p \frac{s + K_i K_p^{-1}}{s} \cdot \frac{N(s)}{D(s)} \quad \text{versus} \quad C(s)H(s) = \frac{C(s)N(s)}{D(s) + R(s)N(s)}$$

$$R(s) = K_d (\exp(-sT) - 1) \quad \text{with} \quad T = -[\arg G(j\omega_0)] \omega_0^{-1}$$

Proposed (plug-in) time-delay based compensator (R) scheme

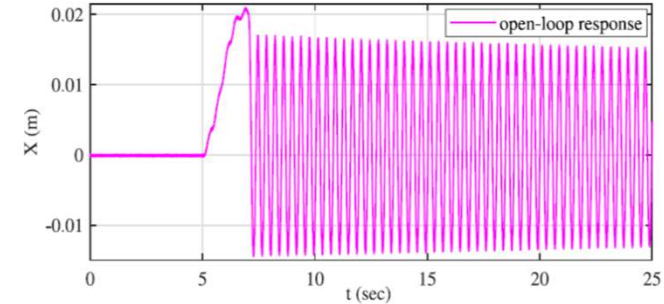


• Low-damped oscillatory non-collocated passive load

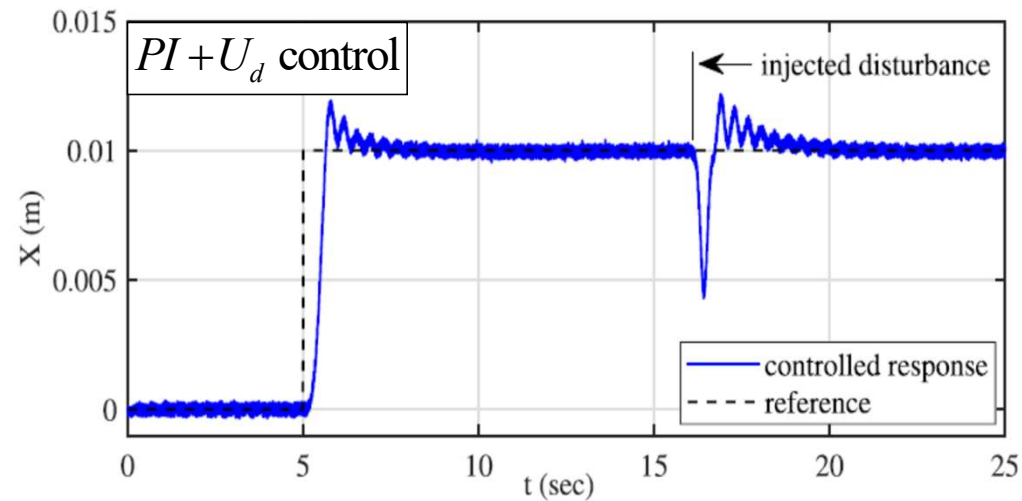
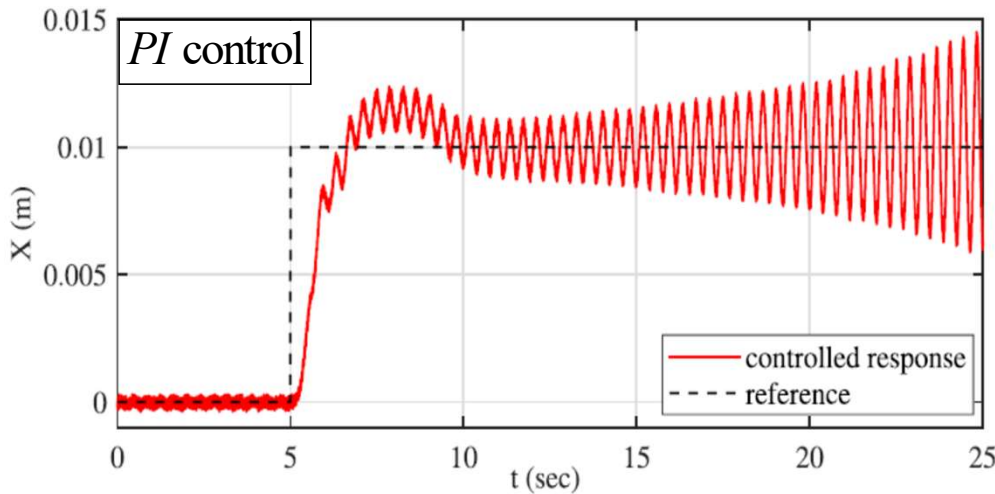


Nominal values of the system parameters.

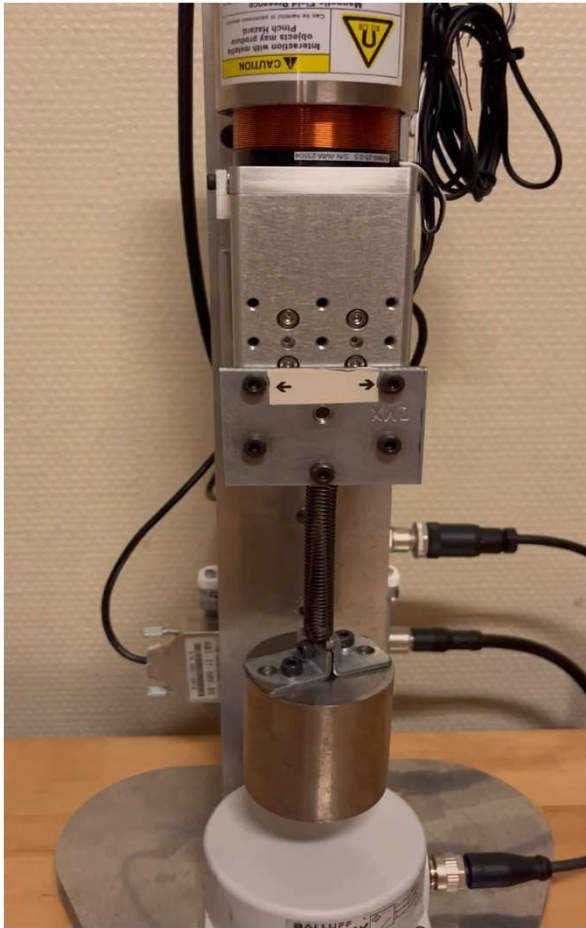
Parameter	Unit	Value	Meaning
m_1	kg	0.6	Actuator mass
m_2	kg	0.75	Load mass
k	N/m	200	Spring constant
σ	kg/s	200	Actuator damping
δ	kg/s	0.01	Spring damping
R	V/A	5.23	Coil resistance
Ψ	V s/m	17.16	EMF constant
g	m/s ²	9.81	Gravity constant



• PI feedback control versus PI + time-delay based compensator (U_d)

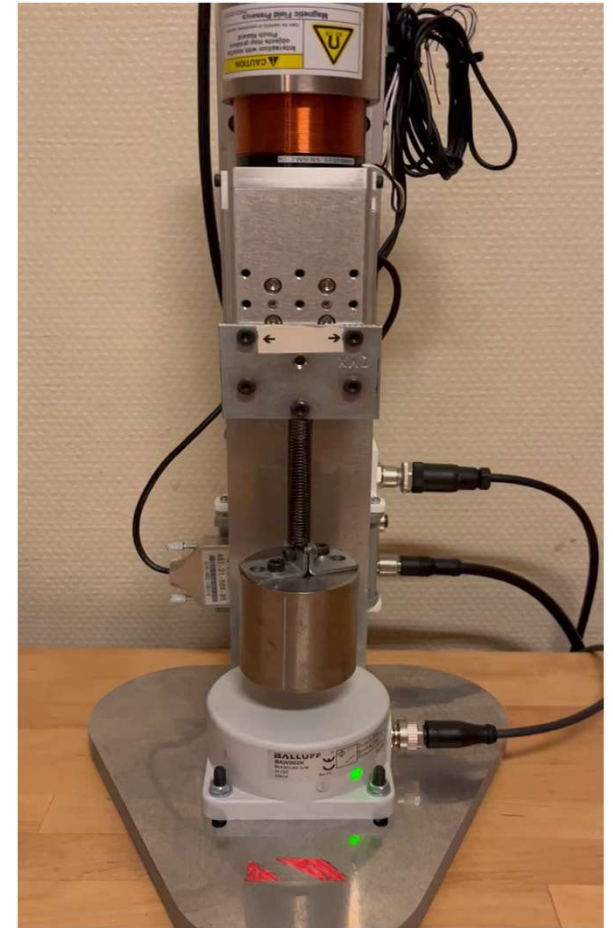
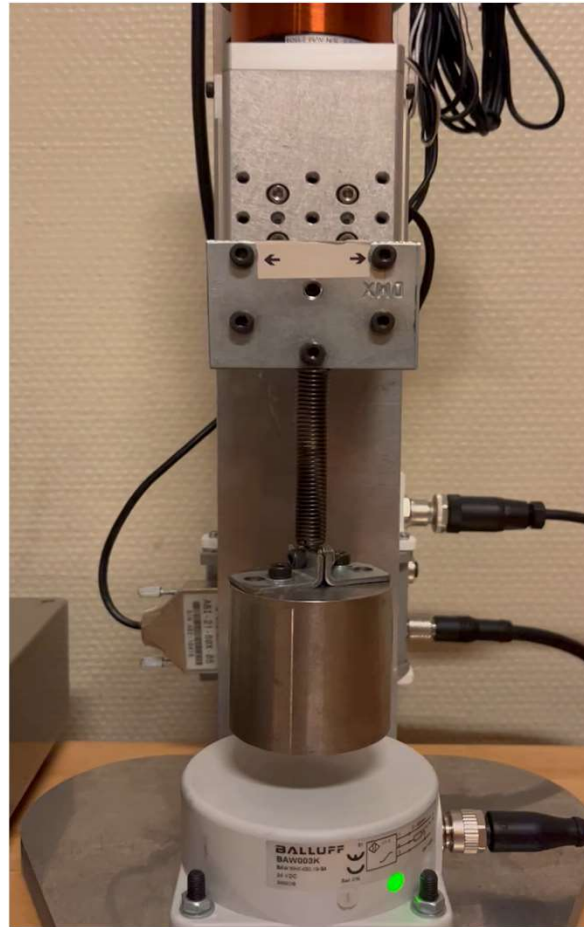


- Free-fall (uncontrolled)



- PI + time-delay based

- PI control

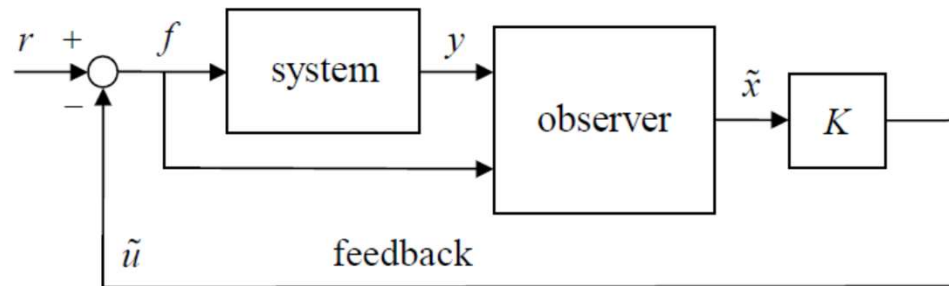
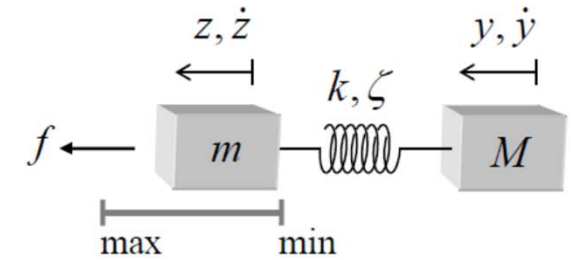


- When using state-feedback with observer

$$x \equiv (x_1, x_2, x_3, x_4)^\top = (\dot{z}, z, \dot{y}, y)^\top$$

$$\begin{aligned} \dot{x} &= Ax + Bf + D, \\ y &= Cx, \end{aligned} \quad \Rightarrow \quad G(s) = C(sI - A)^{-1}B$$

$$e(t) = x(t) - \tilde{x}(t) \quad \Rightarrow \quad \dot{e}(t) = \tilde{A}e(t) = (A - QC)e(t)$$


 \Rightarrow

$$\dot{\tilde{x}} = (A - BK - QC)\tilde{x} + Bf + Qy$$

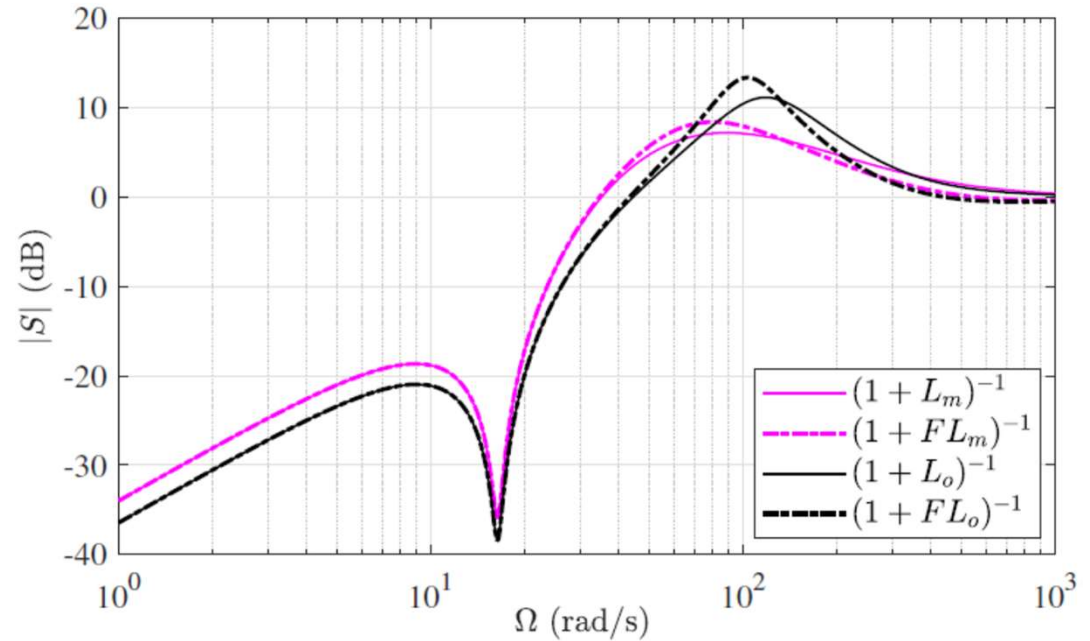
- Consider additional (practical) stability features, see [3]

$$\text{loop transfer function from } r(s) \text{ to } \tilde{u}(s) \quad L_o(s) = K(sI - A + BK + QC)^{-1} [B \ Q] \begin{bmatrix} 1 \\ G(s) \end{bmatrix}$$

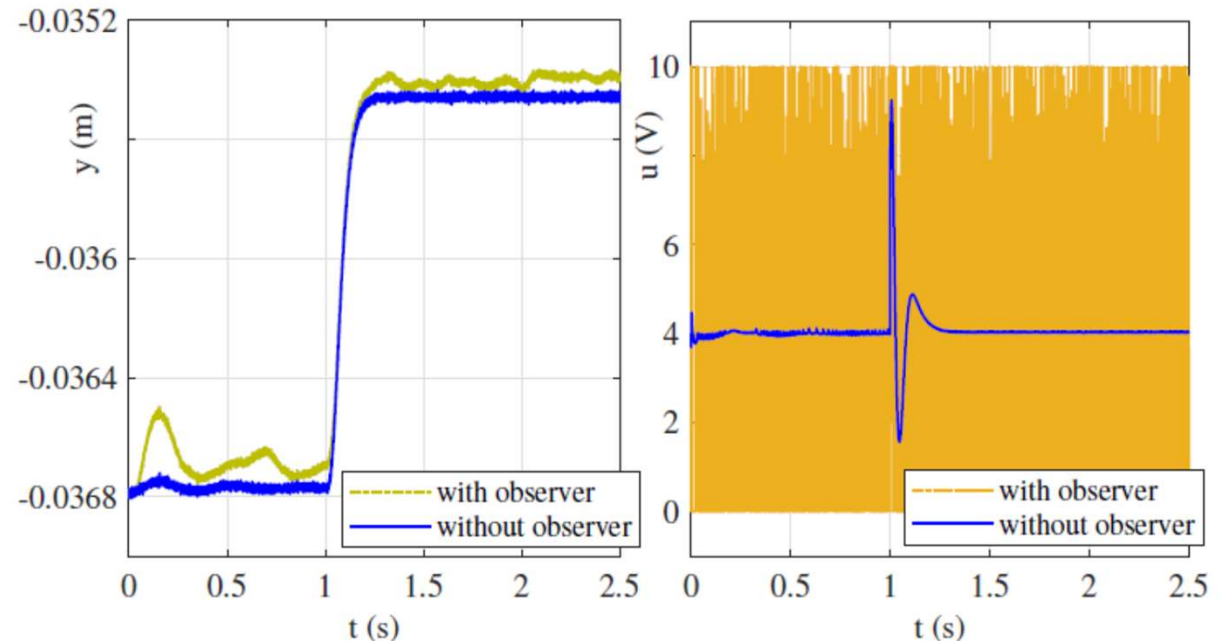
$$L_m(s) = K(sI - A)^{-1}B \quad \Rightarrow$$

$$S_{\max} = \max_{\Omega} |S(i\Omega)| = \max_{\Omega} \left| (1 + L(i\Omega))^{-1} \right|$$

- Low ‘stability margin’ S_{\max} of observer-based feedback



- As consequence – unfeasible control behavior (even in the numerical simulations)



- Time-delay based feedback control [2] can be online tuned (i.e. be adaptive) by the robust frequency estimator [1], as shown in [3]

$$w(t) = y(t) - y\left(t - \frac{\pi}{\beta}\right), \quad \omega < \beta < 3\omega, \quad (17)$$

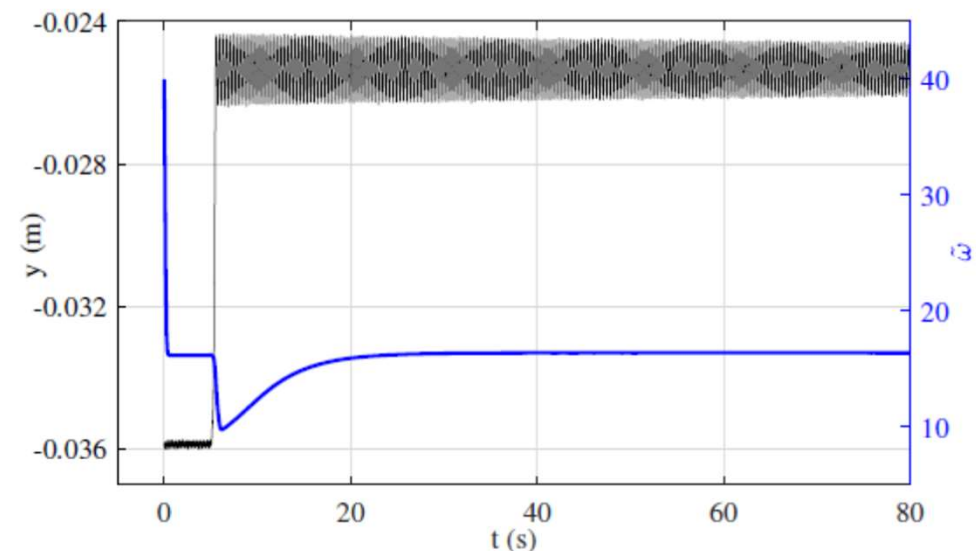
where β is a free adjustable time-delay parameter. Assuming a biased (by Y_0) harmonic oscillation

$$y(t) = Y_0 + Y \sin(\omega t + \phi), \quad (18)$$

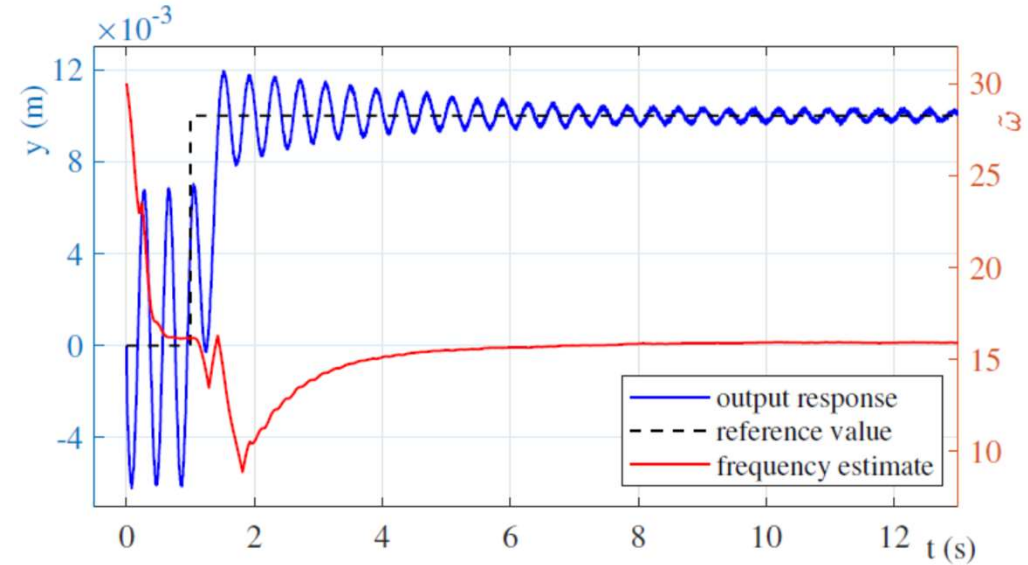
and substituting it into (17) results in

$$w(t) = Y \left(\sin(\omega t + \phi) - \sin\left(\omega t + \phi - \omega \frac{\pi}{\beta}\right) \right). \quad (19)$$

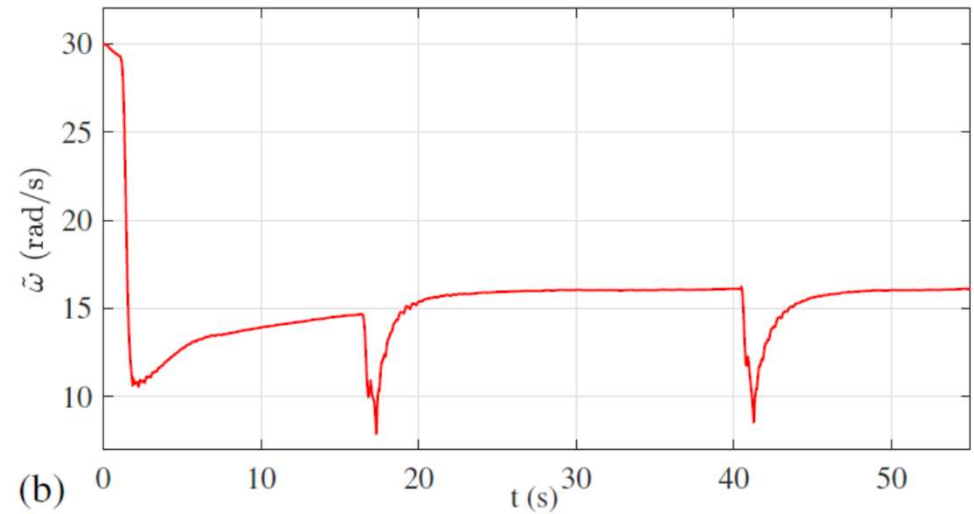
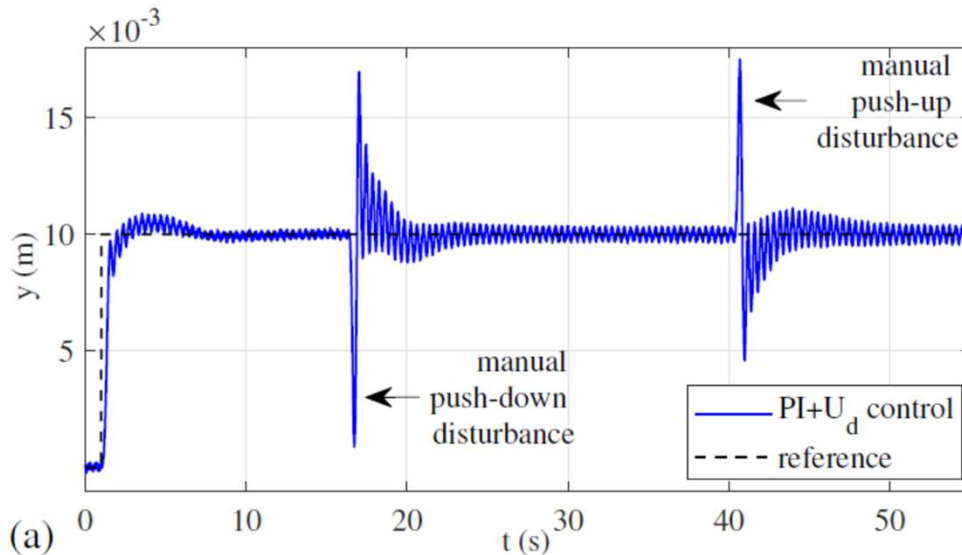
- Converging oscillations frequency, even in presence of stepwise transients (here experiments)



- Experimental evaluation for oscillatory initial conditions



- Experimental evaluation with injected external disturbances, [3]



■ Previous related works and references

1. One-parameter robust global frequency estimator for slow-varying amplitude and noisy oscillations. M Ruderman. *Mechanical Systems and Signal Processing*, 170, pp. 108756, 2022
2. Time-delay based output feedback control of fourth-order oscillatory systems. M Ruderman. *Mechatronics*, 94, pp. 103015, 2023
3. Adaptive time delay based control of non-collocated oscillatory systems. M Ruderman. arXiv:2311.14979, <http://arxiv.org/abs/2311.14979>, 2023, to appear in *IEEE MED*, 2024
4. An adaptive notch filter for frequency estimation of a periodic signal. M. Mojiri, A.R. Bakhshai. *IEEE Transactions on Automatic Control*, 49 (2), pp. 314–318, 2004
5. A globally convergent frequency estimator. L. Hsu, R. Ortega, G. Damm. *IEEE Transactions on Automatic Control*, 44 (4), pp. 698–713, 1999
6. Stability and stabilization of systems with time delay. R. Sipahi, S.I. Niculescu, C.T. Abdallah, W. Michiels, K. Gu. *IEEE Control Systems Magazine*, 31(1), pp.38–65, 2011
7. Robust output feedback control of non-collocated low-damped oscillating load. M. Ruderman. In *IEEE 29th Mediterranean Conference on Control and Automation (MED)*, pp. 639–644, 2021

Thank you for attention